

## **REMARKS**

Reconsideration of the application is requested in view of the above amendments and the following remarks. Claims 6, 9, 12 and 15 have been amended. Support for the amendments to claims 6, 9 and 12 is provided by at least Figures 6A and 6B of the present application. The amendments to claim 15 are supported by at least Figures 4 and 5 of the present application. No new matter has been added.

### **Objections**

A substitute specification including all revisions during prosecution to date is included after the Remarks section. Both a clean version and a marked-up version showing changes is included.

Claim 2 was objected to for an informality. Applicant submits that the limitations of claim 2 are generic to the tube-sided supply and the tube-sided discharge. Therefore, claim 2 properly depends from claim 15.

### **§ 103 Rejections**

Claims 2, 4, 6, 8, 9, 11, 12, and 14-23 were rejected under 35 U.S.C. §103(a) as being unpatentable over Miller (US 2,110,024) in view of Kipp (US 4,452,070). Applicant respectfully traverses this rejection.

Miller '024 discloses a heat exchanger that uses sealing plugs 53 that are removeably inserted in bores of headers 41, 43. The plugs 53 are screw plugs that do not include a clamping member as required by claims 6, 9, 12 and 15.

Kipp discloses an apparatus for testing casings for pressure and tension limits. The purpose and function of the Kipp apparatus is unrelated and nonanalogous to the heat exchanger disclosed by Miller '024. Further, the Kipp apparatus is not a sealing plug as it is configured specifically with a tubing that extends through the plug to permit injection of high-pressure fluid into the casing through the plug. Therefore, one skilled in the art of heat exchangers would not look to the casing plug having a tube there through as disclosed by Kipp for modifying the sealing plugs disclosed by Miller '024.

Miller '024 and Kipp also fail to disclose or suggest that each tube bore of the plate has "a first diameter segment adjacent to the top face, the first diameter segment

having a constant diameter, and a second diameter segment spaced between the back face and the top face, the second diameter segment having a diameter less than the diameter of the first diameter segment," as required by claim 6. The bore disclosed by Miller '024 appears to have a single diameter. The tube bore disclosed by Kipp includes a first section having a diameter that tapers to a second diameter. Therefore, Miller '024 and Kipp fail to disclose or suggest every limitation of claim 6 and the claims that depend from it.

Miller '024 and Kipp also fail to disclose or suggest that "the plugs comprise a body part having a bore formed therein extending along a portion of a length of the body part and a clamping member, wherein the clamping member is coupled to the body part via the bore of the body part," as required by claim 9, or that the body part of the plug includes "an open end and a closed end and the clamping member is attached to the body part in the open end," as required by claim 12. Miller '024 fails to disclose or suggest a plug with a bore or open end. Kipp discloses an apparatus that includes a bore/tube that extends completely through the apparatus between opposing ends. Therefore, Miller '024 and Kipp fail to disclose or suggest every limitation of claims 9 and 12 and the claims that depend from them.

Miller '024 and Kipp further fail to disclose or suggest that "the connecting channel includes a cylindrical cross-section having a diameter greater than a diameter of the tube bores," as required by claim 15. The transverse cavities in headers 41, 43 disclosed by Miller '024 that are connected to the conduits 44, 45 are not cylindrical. Kipp fails to remedy the deficiencies of Miller '024 as it relates to claim 15. Therefore, Miller '024 and Kipp fail to disclose or suggest every limitation of claim 15 and the claims that depend from it.

Claims 2, 4, 9, 11, 15-17, and 20-21 were rejected under 35 U.S.C. §103(a) as being unpatentable over Miller (US 2,608,388) in view of Kipp (US 4,452,070). Applicant respectfully traverses this rejection.

Miller '388 discloses a heat exchanger with headers 1, 2 at opposing ends and threaded plug members 6 screwed into holes in the headers 1, 2. The plugs 6 do not include clamping members as required by claims 9 and 15. Further, the plugs do not

include a bore formed therein, and the headers 1, 2 do not include connecting channels having a cylindrical cross-section.

As described above, the apparatus disclosed by Kipp is unrelated and nonanalogous art to heat exchangers. Therefore, one skilled in the art of heat exchangers would not look to the casing plug having a tube there through as disclosed by Kipp for modifying the sealing plugs disclosed by Miller '388.

Furthermore, Kipp fails to remedy the deficiencies of Miller '388 as it relates to claims 9 and 15. Kipps fails to disclose or suggest plugs that include "a body part having a bore formed therein extending along a portion of a length of the body part and a clamping member, wherein the clamping member is coupled to the body part via the bore of the body part," as required by claim 9. Kipp also fails to disclose or suggest connecting channels that include "a cylindrical cross-section having a diameter greater than a diameter of the tube bores," as required by claim 15. Therefore, Miller '388 and Kipp fail to disclose or suggest every limitation of claim 9 and 15 and the claims that depend from them.

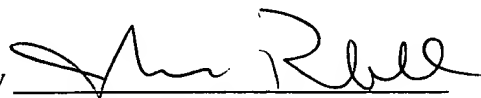
In view of the above, Applicant requests reconsideration of the application in the form of a Notice of Allowance. Any questions or concerns regarding this communication can be directed to the undersigned attorney at (612)371.5387.

Respectfully submitted,

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## Substitute Specification - Marked up Version

Title: Heat exchanger

5 The invention relates to a heat exchanger, comprising a shell designed as a pressure vessel, provided with shell-sided supply and discharge means with which the shell can be flowed through with a first medium under pressure, further comprising a nest of tubes extending at least partly within the shell, provided with tube-sided supply and discharge means with which the tubes from the nest can be flowed through with a  
10 second medium in heat exchanging contact with the first medium under pressure, of which nest the individual tubes are each included with a supply and discharge side in tube bores extending substantially transversely to the plane of a tube plate included in the shell.

Such a heat exchanger is known from practice as a heat exchanger of the "shell  
15 and tube type" and is used for exchanging heat between two mediums under pressure.

With the known heat exchanger, in the event of high pressures of the mediums, the tube-sided supply and discharge means are connected with the tubes via a so-called D-head, as mentioned in the TEMA 8<sup>th</sup> edition, Figs. N1.2, or variations thereon. The D-head comprises a tube plate manufactured from wrought iron, in which tube bores are  
20 provided. The tube plate is provided with an integrated upright circumferential edge, which can be closed with the aid of a detachable cover part to form a central pressure chamber extending along the tube plate. Often, the central pressure chamber is divided into a supply and a discharge part by means of a partition. The supply and discharge means are designed as tubes which are each connected with the central pressure  
25 chamber via a tube-sided supply and discharge opening, provided in the upright circumferential edge.

During use, the nest of tubes is flowed through with the second medium via the tube-sided supply and discharge means. The second medium flows from a supply tube via the tube-sided supply opening into the supply part of the central chamber.  
30 Subsequently, the second medium flows from the supply part of the central chamber in a flow direction extending substantially transversely to the plane of the tube plate via

the tube bores into the tubes. Subsequently, the second medium flows via the tube bores substantially transversely to the plane of the tube plate from the tubes into the discharge part of the central chamber. Then, the second medium flows into a discharge tube via the tube-sided discharge opening through the upright edge.

5           In the above-mentioned case, the nest of tubes is substantially U-shaped from the supply part of the central chamber to the discharge part of the central chamber. The supply and discharge sides of the individual tubes are included in the same tube plate.

          However, the heat exchanger according to the opening paragraph can also comprise two D-heads between which the nest of tubes extends. In such a case, the  
10       supply and discharge sides of the individual tubes from the nest of tubes are included in tube bores of different tube plates; then, two tube plates are included in the shell. The central chamber of the second D-head can function as a connecting chamber, but can also function as a separate outflow chamber, the central chamber in the first D-head only functioning as inflow chamber.

15           A drawback of the known heat exchanger of the shell and tube type is that the D-head has to be of very robust design. In particular, this is the case when the difference between the pressure exerted by the second medium in the central chamber and the pressure exerted by the first medium in the pressure vessel is large. The pressure exerted by the second medium actually results in a large pressure load on the  
20       plane of the tube plate. Further, as a result of the high pressure of the second medium, the cover part and its connection also have to be of very robust design. The robust D-head is heavy, relatively expensive in manufacture and, further, takes up relatively much space. Additionally, to clean the nest of tubes, the cover part has to be detached and placed back, which, considering the pressure-resistant robust sealing of the cover  
25       part, is a time-consuming operation.

          The invention has for its object to provide a heat exchanger of the type mentioned in the opening paragraph, in which the above-mentioned drawbacks are avoided. To that end, the heat exchanger according to the invention is characterized in that the tubes are connected with the tube-sided supply and discharge means via  
30       connecting channels situated in the plane of the tube plate and crossing the tube holes. The use of connecting channels integrated into the tube plate ensures that a central

pressure chamber extending along the plane of the tube plate can be avoided. In particular, the connecting channels integrated into the tube plate can ensure that the resulting pressure load exerted on the tube plate is substantially neutral, so that the construction of the tube plate can be considerably lighter. Furthermore, as the use of  
5 the central pressure chamber extending along the plane of the tube plate is avoided, the upright edge and the cover part of the conventional D-head can be omitted. Thus, the construction can be considerably lighter and can be of simpler design, and the length of the heat exchanger can be reduced.

In an advantageous embodiment, the connecting channels are designed as  
10 straight connecting bores, each crossing at least two tube bores. One connecting bore can cross, for instance, one row of tube bores. One connecting bore can also cross, for instance, parallel rows of tube bores.

In a further advantageous embodiment, the tube bores are designed to be continuous, and the tube bores, at their free ends, are sealed with plugs. This ensures  
15 that the tubes, each individually, can be accessible for cleaning purposes. Preferably, the tube bores are provided with screw thread for detachably receiving a plug provided with corresponding screw thread. Preferably, clamping means are provided for clampingly receiving a sealing ring between a top face of the tube plate and the plug.

The supply and discharge means can be designed as tube ends provided on the  
20 peripheral edge of the tube plate or as radially outwardly extending distribution chambers. Further, the supply and discharge means can be designed as tubes connected with the connecting channels via a tee. Optionally, the connecting bores can be interconnected by means of one or more central connecting channels.

Further advantageous embodiments of the invention are represented in the  
25 subclaims.

The invention will be explained in more detail with reference to an exemplary embodiment represented in a drawing. In the drawing:

Fig. 1 shows a schematic side view of a heat exchanger according to the invention;

30 Fig. 2 shows a schematic front view of the heat exchanger of Fig. 1 from the tube plate;

Fig. 3 shows a schematic top plan view of the tube plate and the nest of tubes of the heat exchanger of Fig. 1;

Fig. 4 shows a longitudinal section of an alternative embodiment of the tube plate;

5 Fig. 5 shows a cross section along the line A-A of the tube plate of Fig. 4;

Fig. 6A shows a detail of a free end of a tube bore of the tube plate of Fig. 4;

Fig. 6B shows a perspective view of a sealing plug for the tube bore of 6A in a taken-apart condition.

10 In the Figures, identical or corresponding parts are designated by the same reference numerals. The Figures are mere schematic representations in elucidation of a preferred embodiment of the invention.

With reference to Fig. 1 – Fig. 3, a heat exchanger 1 is shown. The heat exchanger 1 comprises a shell 2 designed as a pressure vessel. The shell 2 of the heat exchanger is positioned on supports 2A. The shell 2 is provided with shell-sided supply means 3 and with shell-sided discharge means 4 with which the shell 2 can be flowed through with a first medium, for instance water under pressure, for instance at least 3 bar. The heat exchanger 1 further comprises a nest 5 of tubes 6 extending within the shell 2. The nest 5 is provided with tube-sided supply means 7 and tube-sided discharge means 8 with which the tubes 6 from the nest 5 can be flowed through with a second medium, for instance natural gas under pressure, for instance 300 bar, such that the first medium and the second medium are in heat exchanging contact.

20 Depending on the mediums and the use of the heat exchanger, the pressure of the first medium can be chosen to be from, for instance, practically zero (vacuum) to 300 to 400 bar, while the pressure of the second medium can be chosen to be between, for instance, 80 to 700 to 1000 bar.

In the shell 2 of the heat exchanger 1, a tube plate 9 is included. The tube plate 9 is received in the shell 2 of the heat exchanger 1, such that it forms an integral part of the wall. To that end, a tube-shaped end of the wall 2 is screwed onto the tube plate 9 with the aid of a flange 10.

30 The tube plate 9 is designed as a round disc extending in a plane V. The tube plate 9 is provided with a large number of blind tube bores 11 extending transversely to

the plane V. In each tube bore 11, a supply side 6A or a discharge side 6B of a tube 6 is included.

In the tube plate 9, two connecting channels 12A, 12B are provided. The connecting channels 12A, 12B are each designed as a cylindrical bore. The first  
5 connecting channel 12A crosses a first group of tube bores 11A, such that the supply sides 6A of the tubes 6 from the nest 5 included in the tube bores 11 are connected with the tube-sided supply means 7. In a corresponding manner, a second connecting  
channel 12B crosses the tube bores 11B, such that the discharge sides 6B of the tubes 6 from the nest 5 included in the tube bores from that group are connected with the tube-  
10 sided discharge means 8.

The tube-sided supply and discharge means are designed as radially outwardly extending tube ends 14A, 14B of respective supply and discharge conduits, provided on the circumferential edge 13 of the tube plate 9.

The tube bores 11 in the tube plate 9 can be designed to be blind, but can also be  
15 designed to be continuous and are sealed with plugs as will be explained below in more detail with reference to Figures 5 and 6.

Referring to Figs. 4 – 6, it is shown in detail how the tubes 6 with their respective supply sides 6A and discharge sides 6B are included in tube bores 11 of a tube plate 9. The tube bores 11 in the body part of this tube plate are designed to be  
20 continuous and extend from a first end 15 located near a back face 20 of the tube plate 9, in which end the tubes 6 are included, to a second free end 16 located near a top face 19 of the tube plate 9, in which end a plug 17 can be received for sealing purposes. Preferably, the tubes 6 are provided in the tube bores 11 by means of a welded joint 18. Such a welded joint 18 can be provided with the aid of a hole weld extending from the  
25 top face 19 of the tube plate 9 into the tube bore 11. The tubes 6 can also be provided in the tube bores in different manners, for instance by means of clamping, rolling or screw thread. It is noted that the back face 20 of the tube plate 9 forms part of the wall of the shell 2 of the heat exchanger 1.

With reference to Fig. 5, it is shown that the connecting channels 12 are  
30 designed as blind connecting bores 12C-12F, each crossing a row of parallel tube bores 11. As appears from Fig. 4 and 5, the connecting channels and tube bores 11 are, in



each case, surrounded by material of the tube plate 9, so that the pressure exerted by the second fluid can be absorbed on all sides and no resultant pressure force is created on the tube plate 9.

The bores 12C – 12F can be connected via a tee with a supply or a discharge conduit, for instance the bores 12C and 12D via an inlet tee and the bores 12E and 12F via an outlet tee.

The second free ends 16 of the tube bores 11 are provided with screw thread 21, as shown in Fig. 6A.

In the screw thread 21, a plug 17 provided with corresponding screw thread 22 can be detachably received. The plug 17 comprises a body part 23 which can be received in the free end 16 of the tube bore 11. Further, the plug 17 comprises a ~~sealing~~ ring-clamping member 24 which can be screwed onto the body part 23 with the aid of a bolt (not shown), such that a sealing ring 25 and, optionally, a support ring 26 can be sealingly screwed onto the top face 19 of the tube plate 9. A suitable sealing ring is the "variseal" from the firm of Busak + Shambam. By detaching the plug 17, a tube 6 from the nest 5 can be made individually accessibly for cleaning purposes.

It will be clear that the invention is not limited to the embodiments represented here. The heat exchanger can comprise, for instance, two tube plates 9 between which a tube nest 5 is provided in a stretched configuration. Then, the shell can be formed by connecting the back faces of the two tube plates with the aid of a tube. The connecting channels from one tube plate can be coupled to the tube-sided supply means, while the connecting channels from the second tube plate are connected with the tube-sided discharge means. It is, however, also possible to connect a first group of tube bores in the first tube plate, via connecting channels, with the tube-sided supply means and to connect a second group of tube bores in the first tube plate, via other channels, with the tube-sided discharge means, while in the second tube plate the connecting channels form a connection between a first group of tube bores and a second group of tube bores.

Further, it is noted that the tube plate can have another shape, for instance rectangular, square or oval.

Furthermore, the tube bores can be sealed in different manners, for instance with hole welds, and the tubes can be connected in different manners, for instance by

clamping. Additionally, the plugs can be designed differently, for instance as bolts whose heads function as clamping means for clamping a sealing ring against the top face of the tube plate. Further, the connecting channels can be designed to be continuous and be sealed on one side with a plug or hole weld. Additionally, the

5 channels can be curved.

Such variants will be clear to those skilled in the art and are deemed to be within the scope of the invention as defined in the appended claims.

Abstract

A heat exchanger, comprising a shell designed as a pressure vessel, provided with shell-sided supply and discharge means with which the shell can be flowed through with a first medium under pressure. The heat exchanger further comprises a  
5 nest of tubes extending at least partly within the shell, provided with tube-sided supply and discharge means with which the tubes from the nest can be flowed through with a second medium in heat exchanging contact with the first medium under pressure. The individual tubes of the nest are each included with a supply and a discharge side in tube bores extending substantially transversely to the plane of a tube plate included in the  
10 shell. The heat exchanger has the special feature that the tubes are connected with the tube-sided supply and discharge means via connecting channels located in the plane of the tube plate and crossing the tube holes.